

Seat No.: \_\_\_\_\_

Enrolment No. \_\_\_\_\_

**GUJARAT TECHNOLOGICAL UNIVERSITY**

**BE - SEMESTER-V (NEW) - EXAMINATION – SUMMER 2018**

**Subject Code:2151909**

**Date:04/05/2018**

**Subject Name:Heat Transfer**

**Time:02:30 PM to 05:00 PM**

**Total Marks: 70**

**Instructions:**

1. Attempt all questions.
2. Make suitable assumptions wherever necessary.
3. Figures to the right indicate full marks.

	<b>MARKS</b>
<b>Q.1</b> (a) Define following: (1) Thermal Conductivity (2) Prandtl Number (3) Solid angle	<b>03</b>
(b) State significance of critical radius of insulation and derive its equation.	<b>04</b>
(c) Differentiate (1) Convection and Radiation modes of heat transfer. (2) Natural and Force convection heat transfers.	<b>07</b>
<b>Q.2</b> (a) Discuss lumped parameter analysis and state its assumptions.	<b>03</b>
(b) A furnace wall is made up of two layers of thickness 250 mm, 100 mm with thermal conductivity of 1.65 and 9.2 W/m °C respectively .The inside is exposed to gases at 1250°C with a convection coefficient of 25 W/m <sup>2</sup> °C and the inside surface is at 1100 °C, the outside surface is exposed to air at 25 °C with convection coefficient of 12 W/m <sup>2</sup> °C.The overall heat transfer coefficient.	<b>04</b>
(c) Derive equations of temperature distribution and heat transfer for composite cylinder separating two fluids considering heat flow in radial direction.	<b>07</b>
<b>OR</b>	
(c) Write general differential equation of heat transfer for constant area fin. From this obtain temperature distribution and heat transfer equations for fin insulated at the tip.	<b>07</b>
<b>Q.3</b> (a) Show physical significance of Following non-dimensional numbers: Nu (Nusselt Number), Gr (Grashof Number)	<b>03</b>
(b) Explain concept of thermal boundary layer.	<b>04</b>
(c) Calculate the rate of heat loss from human body which may be considered as a vertical cylinder 300mm in diameter and 175mm high in still air at 15 °C. The skin temperature is 35 °C and emissivity at the skin surface is 0.4. Neglect sweating and effect of clothing. The thermo physical properties of air at mean film temperature are: $K = 0.0263 \text{ w/m } ^\circ\text{C}$ $\nu = 15.53 \times 10^{-6} \text{ m}^2/\text{s}$ $pr = 0.7$ for turbulent flow use $Nu = 0.13( Gr \times Pr)^{0.33}$	<b>07</b>

**OR**

- Q.3** (a) Explain mean film temperature and bulk mean temperature. **03**  
(b) A motor cycle cylinder consists of fins having outside diameter 150 mm and total surface area of 0.27 m<sup>2</sup>. Calculate the rate of heat dissipation from cylinder fins when motorcycle is running at 20 m/s speed. The atmospheric air is at 25 °C and average fin surface temperature is 475 °C. the relevant thermo – physical properties at average temperature of 250 °C are:  $K = 0.0427 \text{ w/m}^\circ\text{C}$   
 $\nu = 40.61 \times 10^{-6} \text{ m}^2/\text{s}$   $pr = 0.677$   
for turbulent flow use  $Nu = 0.036 (Re)^{0.8} (Pr)^{0.33}$   
(c) Show by dimensional analysis that for forced convection Nussle number is the function of Reynolds number and Prandtl number. **07**

- Q.4** (a) Prove that for black body  $\lambda_{\max} T = \text{Constant}$ . Where  $\lambda_{\max} =$  wavelength at which spectral emissive power is maximum and  $T =$  absolute temperature of body. **03**  
(b) Justify that a good absorber is also a good emitter for radiation heat transfer. **04**  
(c) What is radiation shield? Show that presence of n number of radiation shields reduces the radiation heat transfer by a factor of (n+1). **07**

**OR**

- Q.4** (a) Define : Shape factor, Irradiation, Emissivity **03**  
(b) The sun emits maximum radiation at  $\lambda = 0.52 \mu\text{m}$ . Assuming sun to be a black body, calculate the surface temperature of sun and the total emissive power of the sun's surface at that temperature. **04**  
(c) Define Intensity of radiation. Derive relationship between normal intensity and black body total emissive power. **07**
- Q.5** (a) What is condensation? How does drop wise condensation differ from film wise condensation? **03**  
(b) Discuss the various regimes in boiling. **04**  
(c) Derive an expression for log mean temperature difference of counter flow heat exchanger. **07**

**OR**

- Q.5** (a) With neat sketch explain classification of heat exchangers based on relative motion of fluids. **03**  
(b) What is the fouling factor? Explain their effect in Heat Exchanger design. **04**  
(c) A counter flow heat exchanger is employed to cool oil of specific heat  $C_p = 2.45 \text{ KJ/Kg}^\circ\text{C}$  with mass flow rate of 0.55 Kg/sec from 115°C to 40 °C by water. The inlet and outlet temperature of cooling water are 15 °C and 75 °C respectively. The overall heat transfer co-efficient is  $1450 \text{ W/m}^2 \text{ }^\circ\text{C}$ . Using NTU method, calculate: (i) The mass flow rate of water (ii) The effectiveness of heat exchanger (iii) The surface area required. **07**

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