

# Temperature distribution for fins of different configurations

Case	Tip Condition	Temp. Distribution	Fin heat transfer
1	Infinitely long fin $\theta(L)=0$	$e^{-mx}$	$M \theta_0$
2	Adiabatic $(d\theta/dx)_{x=L}=0$	$\frac{\cosh m(L-x)}{\cosh mL}$	$M\theta_0 \tanh mL$
3	Convection heat transfer: $h\theta(L)=-k(d\theta/dx)_{x=L}$	$\frac{\cosh m(L-x) + (\frac{h}{mk}) \sinh m(L-x)}{\cosh mL + (\frac{h}{mk}) \sinh mL}$	$M\theta_0 \frac{\sinh mL + (\frac{h}{mk}) \cosh mL}{\cosh mL + (\frac{h}{mk}) \sinh mL}$

$$\theta \equiv T - T_\infty, \quad m^2 \equiv \frac{hP}{kA_C}$$

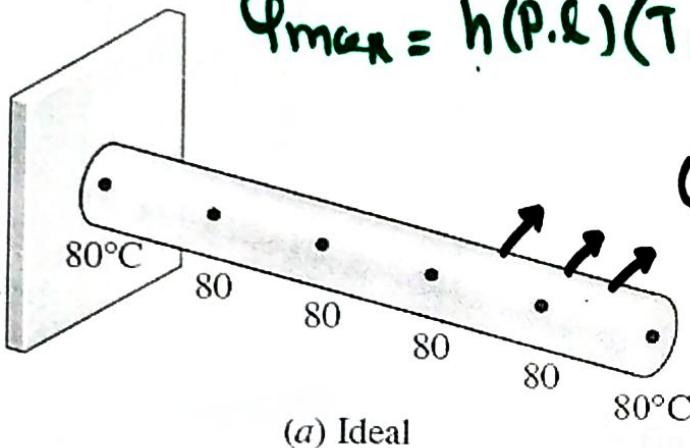
$$\theta_b = \theta(0) = T_b - T_\infty, \quad M = \sqrt{hPkA_C} \theta_b$$

## Fin Efficiency

$$\eta_{\text{fin}} = \frac{\Phi_{\text{fin}}}{Q_{\text{max}}} = \frac{\Phi_{\text{fin}}}{\Phi_{\text{fin}}(k \rightarrow \infty)}$$

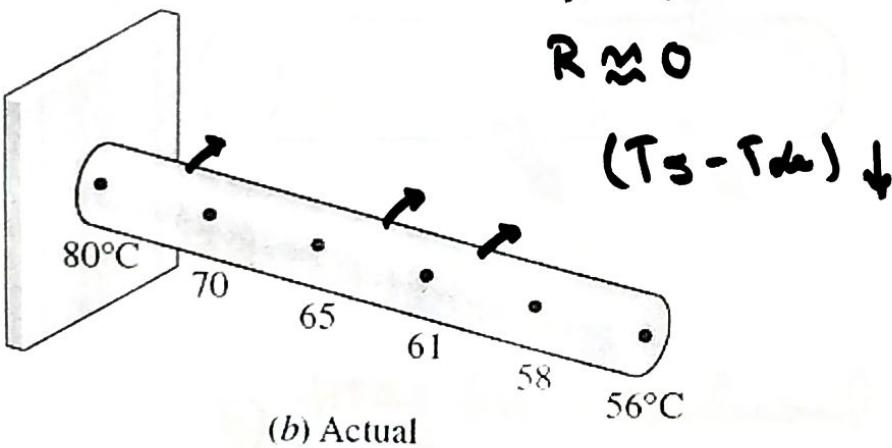
$$\Phi = hA \Delta T$$

$$Q_{\text{max}} = h(P \cdot L)(T_0 - T_K)$$



(a) Ideal

$$(T_s - T_K) = C$$



(b) Actual

$$K \rightarrow \infty$$

$$R \approx 0$$

$$(T_s - T_K) \downarrow$$

$$\boxed{\Phi_{\text{fin}} = \frac{\tan h(mL)}{mL}}$$

a) infinitely long fin ( $L \rightarrow \infty$ )

$$\eta_{\text{fin}} = \frac{\Phi_{\text{fin}}}{Q_{\text{max}}} = \frac{\sqrt{hPAcS} K (T_0 - T_K)}{hP \cdot L (T_0 - T_K)}$$

$$= \sqrt{\frac{hP \cdot AcS K}{h^2 P^2 L^2}}$$

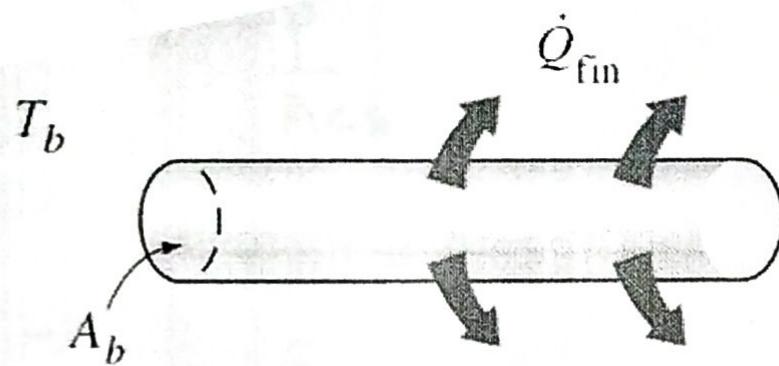
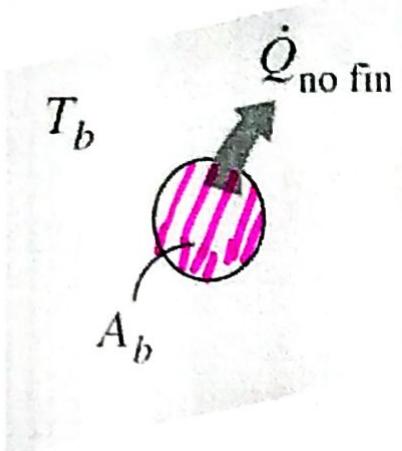
$$\boxed{\eta_{\text{fin}} = \frac{1}{ml}}$$

b) tip of fin insulated

$$\Phi_{\text{fin}} = \frac{\sqrt{hPAcS} K (T_0 - T_K) \tanh ml}{h \cdot (P \cdot L) (T_0 - T_K)}$$

## Fin Effectiveness

$$\epsilon_{fin} = \frac{Q_{with\ fin}}{Q_{without\ fin}}$$



$$Q_{without\ fin} = h A_{cs} (T_0 - T_\infty)$$

a) fin with infinitely long

$$\epsilon_{fin} = \frac{Q_{fin}}{Q_{without\ fin}}$$

$$\epsilon_{fin} = \frac{\sqrt{h P A_{cs} K} (T_0 - T_\infty)}{h A_{cs} (T_0 - T_\infty)}$$

$$= \sqrt{\frac{P \cdot K}{h \cdot A_{cs}}} \quad - \textcircled{I}$$

b) long fin insulated

$$\epsilon_{fin} = \frac{\sqrt{h P A_{cs} K} (T_0 - T_\infty) \tanh m l}{h A_{cs} (T_0 - T_\infty)} = \sqrt{\frac{P \cdot K}{h \cdot A_{cs}}} \tanh m l$$

$$\epsilon_{fin} = \sqrt{\frac{P \cdot K}{h \cdot A_{cs}}}$$

1)  $K$  = Thermal Conductivity is high

2)  $h$  is small

$h \rightarrow$  Natural  
 $h \rightarrow$  forced convection }

3)  $\frac{P}{A_{cs}}$



$$\epsilon_{fin} = 1$$

$$\epsilon_{fin} > 2 \quad ml > 5$$