

Transient Heat Conduction.

$$T = f(x, y, z)$$

$$T = f(x, y, z, t)$$

Application of Transient heat conduction.

1) heating & cooling of metal billets

2) cooling of IC Engine

3) Heat Treatment of metal by quenching

→ The analysis of unsteady state problem of Heat Transfer

1) Analytical method

2) Graphical method

3) Numerical Technique

→ a) Periodic variation

b) Non Periodic variation

Lumped Parameter Analysis ($k \rightarrow \infty$)

solid k



internal Resistance

$$R_{th} = \frac{x}{Ak}$$

air, water



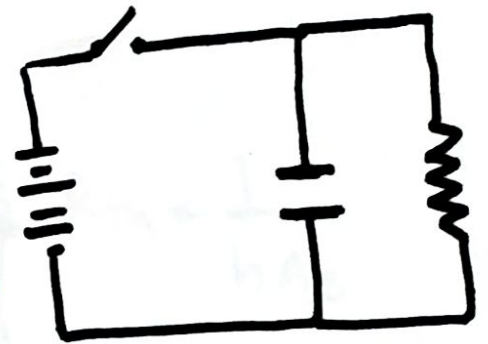
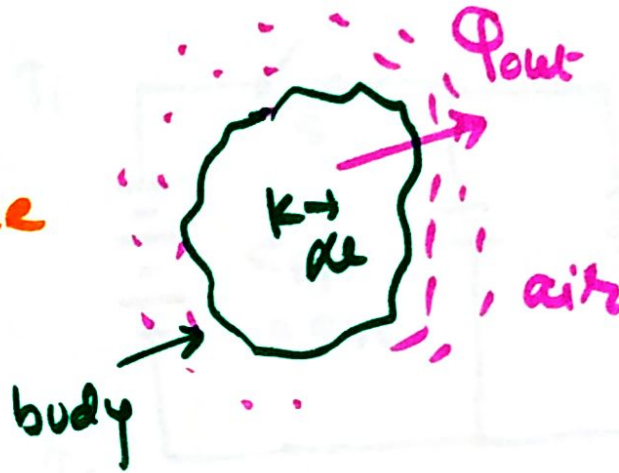
$R_{th} = \text{internal Resistance} = 0$

$\frac{1}{hA} = \text{Convective Resistance}$



- 0 \rightarrow 100
- 1 \rightarrow 95
- 2 \rightarrow 90
- 3 \rightarrow 85

entire body \rightarrow lump



$$z = 0 \quad T = T_i$$

$$z > 0 \quad T = f(z)$$

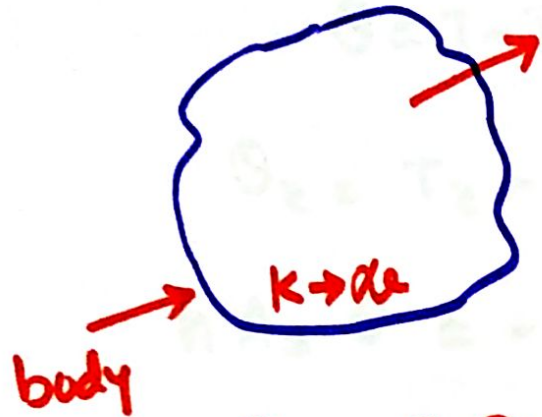
Energy Balance eqⁿ.

Rate of heat flow out the solid

= Rate of heat transfer by convection at surface

$$\dot{Q} = \rho V c \frac{dT}{dt} = h A (T - T_a)$$

Lumped Parameter Analysis

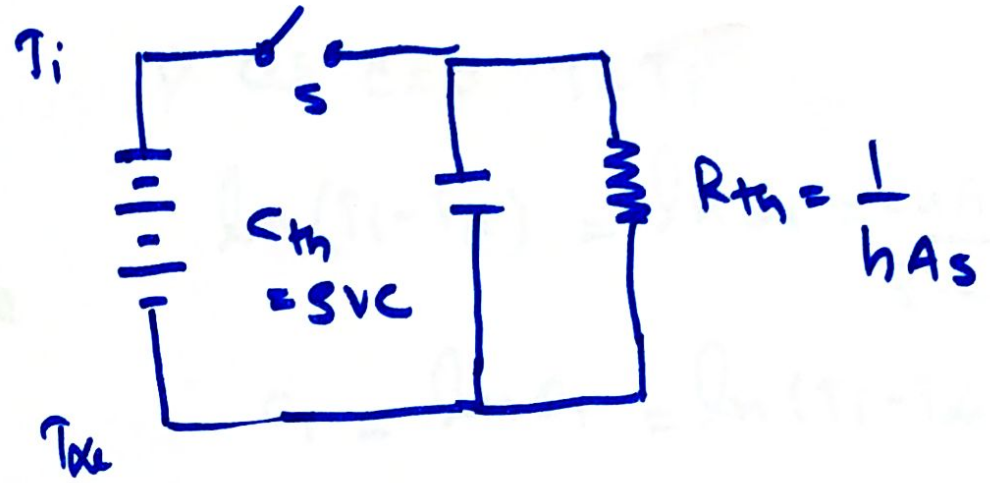


$$z=0 \quad T=T_i$$

Energy Balance eqⁿ

The Rate of Decrease in internal energy of body = The Rate of heat flow at boundary of solid

$$- \rho V C \frac{\partial T}{\partial t} = h A_s (T - T_\infty)$$



$$h A_s (T - T_\infty) = -SVC \frac{\partial T}{\partial z}$$

$$\theta = T - T_\infty$$

$$\theta_z = T_z - T_\infty \quad \frac{d\theta}{dz} = \frac{dT}{dz}$$

$$h A_s \theta = -SVC \frac{d\theta}{dz}$$

$$\frac{d\theta}{\theta} = -\frac{h A_s}{SVC} dz$$

integrating both side

$$\int \frac{d\theta}{\theta} = \int -\frac{h A_s}{SVC} dz$$

$$\ln \theta = -\frac{h A_s}{SVC} z + C_1$$

*> Boundary Condition.

↳ at $z=0$ $T=T_i$

$$\ln(T_i - T_\infty) = \ln \theta_i = -\frac{h A_s (0) + C_1}{SVC}$$

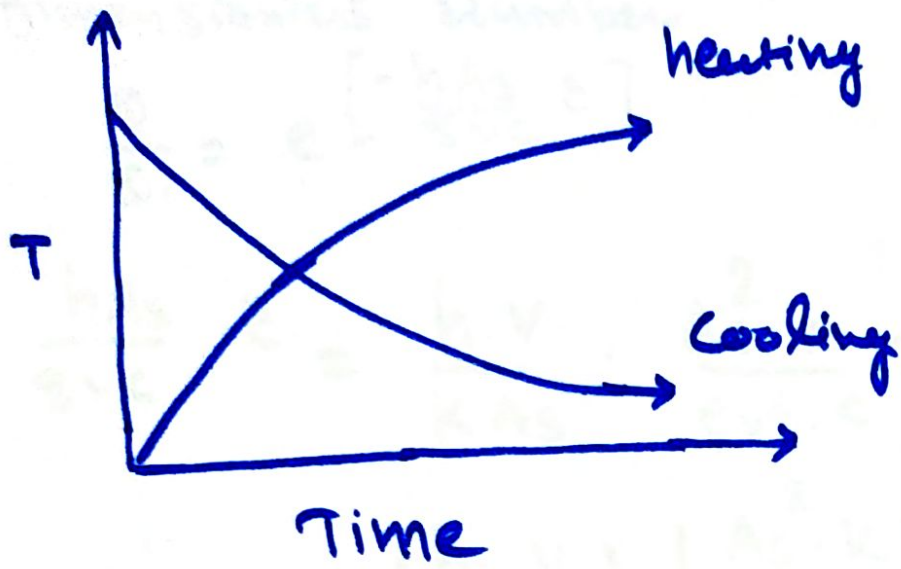
$$C_1 = \ln \theta_i = \ln(T_i - T_\infty)$$

$$\ln \theta = -\frac{h A_s}{SVC} z + \ln \theta_i$$

$$\ln \theta - \ln \theta_i = -\frac{h A_s}{SVC} z$$

$$\ln \left(\frac{\theta}{\theta_i} \right) = -\frac{h A_s}{SVC} z$$

$$\frac{\theta}{\theta_i} = e^{\left[-\frac{h A_s}{SVC} z \right]}$$



$\frac{SVC}{hAs}$ = Thermal time constant

$$\tau_{th} = \frac{SVC}{hAs} = \frac{1}{hAs} \cdot SVC$$

$$\tau_{th} = R_{th} \cdot C_{th}$$

$B_i = \frac{\text{Directed distance of solid}}{\text{Convective resistance at surface}}$

$$= \frac{Lc}{hAs} \times \frac{1}{\Delta T}$$

$$B_i = \frac{hLc}{k}$$

For lumped parameter analysis

$$B_i \leq 0.1$$

Fourier Number

$$Fo = \frac{kAs \Delta T / L}{SVC \Delta T / 2} = \frac{kAs \cdot 2}{SAs c \cdot L}$$

$$= \frac{k}{Lc} \cdot \frac{2}{L} = \frac{kT}{L^2}$$

Dimensionless Number

$$\frac{\theta}{\theta_i} = e \left[-\frac{h A_s z}{\rho v c} \right]$$

$$\frac{h A_s}{\rho v c} \cdot z = \frac{h \cdot V}{K A_s} \cdot \frac{A_s^2 \cdot K}{\rho v^2 \cdot c} \cdot z$$

$$= \left(\frac{h}{K} \cdot \frac{V}{A_s} \right) \left(\frac{A_s \cdot K}{\rho v^2 \cdot c} \right) z$$

$$\alpha = \frac{K}{\rho c}$$

$L_c = \text{characteristic length}$
 $= \frac{V}{A_s}$

$$= \left(\frac{h \cdot L_c}{K} \right) \cdot \left(\frac{\alpha \cdot z}{L_c^2} \right)$$

$$\text{BIOT Number} = Bi = \frac{h \cdot L_c}{K}$$

$$\text{FOURIER Number} = Fo = \frac{\alpha z}{L_c^2}$$

1) BIOT Number

$Bi = \frac{\text{Internal Resistance of Solid}}{\text{Convective Resistance at Surface}}$

$$= \frac{L_c}{A_s K} \times \frac{1}{1/h A_s}$$

$$Bi = \frac{h \cdot L_c}{K}$$

⇒ For Jump Parameter Analysis

$$Bi \leq 0.1$$

2) FOURIER Number

$$Fo = \frac{K A_s \Delta T / L_c}{\rho v c \Delta T / z} = \frac{K \cdot A_s \cdot z}{\rho A_s L_c L_c}$$

$$= \frac{K}{\rho c} \cdot \frac{z}{L_c^2} = \frac{\alpha z}{L_c^2}$$

→ Characteristic Length.

$$L_c = \frac{V}{A_s}$$

(a) Plate $L_c = \frac{V}{A_s} = \frac{LBH}{2BH} = L/2$

L, B, H

(b) Cylinder $L_c = \frac{V}{A_s} = \frac{\pi R^3 L}{2\pi RL} = R/2$

(c) Sphere $L_c = \frac{V}{A_s} = \frac{4/3 \pi r^3}{4\pi r^2} = r/3$

(d) Cube $L_c = \frac{l^3}{6l^2} = l/6$